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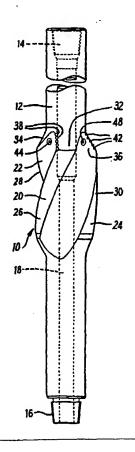
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(54) Title: WELL DRILLING TOOLS

(57) Abstract

A back-reaming stabiliser (10) for incorporation in a bottom-hole assembly (BHA) on the downhole end of a drillstring. The stabiliser consists of a conventional fixed-blade stabiliser having added to it rows of PDCs (38, 40, 42) or other suitable hard inserts along the upper edges of the blades (20, 22, 24) to act as backreaming cutters, nozzles (44, 46, 48) for directing mud at these cutters during backreaming, and a valve (50) normally closing these nozzles. The valve is opened by remote actuation immediately prior to the start of back-reaming. The valve is preferably a sleeve (50) normally held over the nozzles by shear pins (58) which are ruptured by a flow-flocking drop member (60). The drop member preferably has a through bore (76, 78) initially blocked by a burstable diaphragm (80) such that after the sleeve (50) is opened by the drop member, mud pressure can be increased to burst the diaphragm (80) and re-open a passage for mud to the downhole end of the bottom-hole assembly. This divides mud flow between an upflow past the outside of the stabiliser, and mud jets through the nozzles to the back-reaming cutters. The invention avoids the wastage of mud that would occur if the back-reaming cutter nozzles were permanently open.



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"Well Drilling Tools" 1 2 3 This invention relates to well-drilling tools, and relates more particularly but not exclusively to a 4 5 well-drilling tool in the form of a back-reaming 6 stabiliser incorporating back-reaming cutter means and 7 valve means controllably operable to divert 8 well-drilling fluid to said cutter means. 9 10 The back-reaming stabiliser is particularly but not 11 exclusively applicable to the drilling of wells where the drilled formation may swell or slough into the 12 wellbore and thereby restrict egress of the 13 14 drillstring. In this context, it is known that the 15 drilling of wells, particularly oil wells, through geological formations which are under extreme local 16 pressures has inherent problems due to the wellbore 17 18 becoming restricted by these rocks swelling or sloughing into it. In certain instances the formation 19 will exhibit a plastic flow and may restrict the 20 wellbore diameter over many metres. 21 22 23 To counteract this problem it has been common practice to use the uppermost drillstring stabiliser (furthest 24

1	from bit) in the BHA (bottom hole assembly) as a
2	back-reaming stabiliser if a restriction is encountered
3	during withdrawal of the drillstring from the wellbore.
4	In such an instance the drillstring is withdrawn slowly
5	from the wellbore whilst rotating at normal drilling
6	speeds, such that the stabiliser blades cut or ream a
7	passage back through the restricted zone. However,
8	this has the disadvantage that the use of a drillstring
9	stabilising element not specifically designed for a
LO	cutting operation is:-
L1	•
L2	(a) a slow and laborious, and therefore expensive
13	operation; and
14	
15	(b) the stabiliser is usually seriously damaged as a
16	consequence.
L7	
18	A stabiliser which is specifically designed to have an
19	additional back-reaming function by the provision of
20	suitably located cutters will usually require that the
21	cutters be supplied with mud or other appropriate
22	well-drilling fluid for the purposes of cooling,
23	lubrication, and debris removal. The mud will be
24	supplied to the cutters by way of nozzles in the
25	stabiliser. The same mud or other well-drilling fluid
26	will be similarly supplied to the drillbit normally
27	present at the downhole (bottom) end of the BHA.
28	However, the drillbit and the back-reaming cutters will
29	be required to work at mutually different times,
30	according to whether the intended direction of progress
31	of the drillstring is downhole or uphole. Accordingly,
32	the simultaneous supply of mud to the drillbit and to
33	the back-reaming cutters represents an inefficient use
34	of the mud supply, the inefficiency manifesting itself,
35	for example, as an unnecessarily high power consumption

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by the mud pump. Nevertheless, the back-reaming

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2 cutters must be reliably supplied with mud or other 3 appropriate well-drilling fluid during back-reaming 4 operation. 5 It is therefore an object of the invention to provide 6 7 an improved back-reaming stabiliser. 8 9 According to the present invention there is provided a back-reaming stabiliser comprising a tubular body from 10 11 which a plurality of stabiliser blades extend, said 12 tubular body having a through passage for hydraulic 13 fluid to flow internally through said tubular body 14 between opposite ends of said stabiliser in use 15 thereof, said stabiliser blades extending radially 16 outwards of said tubular body, said stabiliser blades 17 extending longitudinally at least partially along said 18 tubular body, said stabiliser blades extending circumferentially at least partially around said 19 20 tubular body, radially outer edges of said stabiliser 21 blades being formed as wellbore-bearing surfaces to 22 provide a radially supportive function in use of said 23 stabiliser, longitudinally common end edges of said 24 stabiliser blades being formed as back-reaming cutter means, and fluid vent means coupling said through 25 26 passage to the exterior of said stabiliser for the flow 27 therethrough of hydraulic fluid from said through 28 passage to said cutter means, said back-reaming 29 stabiliser being characterised by further comprising 30 valve means normally closing said fluid vent means to the flow of hydraulic fluid therethrough, said valve 31 32 means being selectively operable during downhole use of said stabiliser to open said fluid vent means to the 33 34 flow of hydraulic fluid therethrough. 35

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1	Said valve means may comprise a sleeve means movable in
2	said through passage between a first position of said
3	sleeve means in which said sleeve means closes said
4	fluid vent to the passage of hydraulic fluid
5	therethrough, and a second position of said sleeve
6	means in which said sleeve means opens said fluid vent
7	means to the passage of hydraulic fluid therethrough,
8	said sleeve means being movable from said first
9	position thereof to said second position thereof by the
10	application of a substantial pressure differential
11	between longitudinally opposite ends thereof. Said
12	sleeve means is preferably normally retained in said
13	first position thereof by shear means rupturable by
14	said application of said substantial pressure
15	differential between longitudinally opposite ends of
16	said sleeve means thereupon to release said sleeve
17	means from said first position thereof for movement to
18	said second position thereof. Movement of said sleeve
19	means from said first position thereof to said second
20	position thereof may be by way of sliding movement
21	longitudinally along said through passage. Said sleeve
22	means may be formed with an internal passage extending
23	longitudinally therethrough and locally defining said
24	through passage of said stabiliser, said substantial
25	pressure differential being created by causing an at
26	least temporary blockage of flow of hydraulic fluid
27	through said internal passage extending longitudinally
28	through said sleeve means. Such hydraulic flow
29	blockage may be caused by manoeuvring a
30	flow-blockage-inducing means into said internal passage
31	through said sleeve means. Said flow-blockage-inducing
32	means may comprise a drop means introduced to said
33	stabiliser by being dropped into the bore of the
34	drillstring of which the stabiliser forms part. Said

drop means preferably comprises a generally tubular

1	member having a longitudinal through bore initially
2	blocked to the passage of hydraulic fluid therethrough
3	by a rupturable diaphragm extending thereacross, said
4	tubular member being formed to lodge in said sleeve
5	means, preferably by forming said tubular member with
6	an external shoulder dimensioned to engage positively
7	with said sleeve means which may be formed with an
8	internal shoulder for that purpose.
9	
10	Said longitudinally common end edges of said stabilise
11	may be formed as said back-reaming cutter means by
12	embedding a plurality of hard inserts in leading
13	portions of said edges, preferably so as to lie
14	substantially in a common notional surface of
15	revolution, which surface may be conical. Each said
16	hard insert may be a PDC (polycrystalline diamond
17	compact), or a chip of tungsten carbide. Said fluid
18	vent means may comprise a respective fluid nozzle
19	adjacent each said cutter-formed end edge of said
20	stabiliser blades.
21	
22	Embodiments of the invention will now be described by
23	way of example with reference to the accompanying
- 24	drawings wherein:-
25	
26	Fig 1 is an elevation of an embodiment of
27	back-reaming stabiliser in accordance with the
28	invention;
29	
30	Fig 2 is a plan view of the stabiliser of Fig 1;
31	
32	Fig 3 is a longitudinal section of the stabiliser
33	of Fig 1, taken on the line III-III in Fig 2;
34	·
35	Fig 4 is a transverse section of the stabiliser of

1	Fig 1, taken on the line IV-IV in Fig 3; and
2	
3	Fig 5 is a longitudinal section, to a
4	much-enlarged scale, of a drop member for use with
5	the stabiliser of Fig 1.
6	•
7	Referring first to Figs 1-4 (which are different views
8	of the same article, to a common scale), a back-reaming
9	stabiliser 10 in accordance with the present invention
10	is basically formed as a conventional fixed-blade
11	stabiliser, though with certain modifications and
12	additions (detailed subsequently) to meet the object of
13	the invention.
14	
15	The stabiliser 10 comprises an elongate tubular body 12
16	having a standard API (American Petroleum Institute)
17	box connector 14 at it upper end, and a standard API
18	pin connector 16 at its lower end. The connectors 14
19	and 16 enable the stabiliser 10 to be coupled into a
20	BHA (bottom hole assembly; not shown), the BHA being
21	connected to the downhole end of a drillstring (not
22	shown).
23	
24	The tubular body 12 has a through passage 18 for
25	carrying a flow of mud (well-drilling fluid) or other
26	hydraulic fluid between the connectors 14 and 16 in use
27	of the stabiliser 10.
28	
29	In a known manner, the body 12 is integrally formed
30	with three stabiliser blades 20, 22 and 24 each
31	extending radially outwards of the body 12 along a
32	central part of its length. The blades 20-24 are
33	equi-angularly located around the body 12 (see Figs 2
34	and 4), and are spirally shaped (see Fig 1) such that,
35	as a whole, the blades 20-24 extend circumferentially

around the body 12 to define the outer periphery of the 1 2 stabiliser 10 (see Fig. 2). The respective radially 3 outer edges 26, 28 and 30 of the stabiliser blades 20, 22 and 24 are formed as wellbore-bearing surfaces such 4 that the stabiliser 10 provides the conventional 5 6 radially supportive function in operation as part of a 7 BHA. 8 9 The respective upper ends 32, 34 and 36 of the 10 stabiliser blades 20, 22 and 24 are formed as 11 back-reaming cutters by being each inset with a 12 respective row of PDCs 38, 40 and 42 which collectively sit on a notional cone coaxial with the stabiliser 10 13 and converging upwardly. The body 12 is fitted with 14 15 three radially directed nozzles 44, 46 and 38 between each of the cutter sets 38-42 (see Figs 2, 3 and 4). 16 The nozzles 44-48 are each fed from the central through 17 18 passage 18 during back-reaming, but during normal 19 downwardly directed drilling, the nozzles 44-48 are closed off by a sleeve 50 (Figs 3 and 4) which is a 20 21 sliding fit in the bore of the through passage 18. sleeve 50 is held in a position in which it normally 22 23 blocks fluid flow to the nozzles 44-48 by being secured 24 to the lower end of a tubular mounting 52 having an 25 external shoulder 54 which sits on a matching shoulder 56 in the bore of the through passage 18. The sleeve 26 50 is secured to the mounting 52 by means of shear pins 27 28 58 (Fig 3; only one pin being visible). 29 30 When the stabiliser 10 is required to operate as a 31 back-reamer, it is necessary to initiate flow of well-drilling fluid outwards through the nozzles 44, 46 32 and 48 for the reasons previously discussed. 33 34 Consequently, it is necessary to slide the sleeve 50 35 down the bore of the through passage to a position in

1 which the sleeve 50 no longer blocks fluid outflow 2 through the nozzle 44-48. Since the stabiliser 10 will 3 at that time be part of a BHA deep down a well, remote operation of the sleeve 50 is clearly necessary (ie 4 5 direct manual movement of the sleeve 50 in such circumstances is utterly impossible). A drop member 60 6 7 for achieving such remote movement of the sleeve 50 8 will now be detailed with reference to Fig 5 (wherein 9 the drop member 60 is depicted to a much larger scale 10 than the stabiliser 10 is shown in Figs 1-4). 11 12 Referring to Fig 5, the drop member 60 is generally tubular in shape and has an external diameter 13 14 sufficiently small as to allow the member 60 to drop 15 down the bore of the drillstring, and to enter the bore of the sleeve mounting 52 (Fig 3). The upper end of 16 the drop member 60 is formed with a small external 17 18 shoulder 62 dimensioned to seat on an internal shoulder 64 at the lower end of the sleeve 50 (Fig 3). 19 external diameter of the drop member 60 below the 20 21 shoulder 62 is marginally less than the internal diameter of the sleeve shoulder 64 so as to allow the 22 23 drop member 60 to pass down through the sleeve 50 until 24 the shoulder 62 seats on the sleeve shoulder 64, whereupon all further downward movement of the drop 25 member 60 is halted. 26 27 28 The drop member 60 is formed of an upper component 66 29 and a lower component 68 which are normally mutually 30 secured by means of a screw thread 70, a tight threaded connection of the components 66 and 68 conveniently 31 being achieved by the application of spanners (not 32 33 shown) to external flats 72 and 74. The components 66 and 68 have respective longitudinal through bores 76 34 and 78 which are initially sealed off one from the 35

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other by means of a burstable diaphragm 80 clamped 2 between the components 66 and 68 to extend fully across 3 the bores 76 and 78. 5 Operation of the invention will now be described. As described above with reference to Figs 1-4, the 6 7 stabiliser 10 provides the conventional radially 8 supportive function of the conventional fixed-blade 9 stabiliser which it resembles, and which need not be 10 further detailed. When required to function as a back-reamer, normal rotation of the stabiliser 10 is 11 continued (clockwise as viewed from above, ie clockwise 12 13 as viewed in Figs 2 and 4), but a suitable upforce is 14 applied to the drillstring and through it, the upforce 15 is applied to the BHA of which the stabiliser 10 is a 16 part. This causes the cutter sets 42, 44 and 46 to 17 bite into the wellbore intrusion. As already detailed, 18 the nozzles 44, 46 and 48 have to be opened prior to commencement of back-reaming. Between the termination 19 of down-drilling and the commencement of back-reaming, 20 the drop member 60 is released into the bore of the 21 drillstring to drop down or be forced by pumped mud to 22 the stabiliser 10 wherein it passes down the through 23 bores of the sleeve mounting 52 and of the sleeve 50, 24 25 until the external drop member shoulder 62 lodges on 26 the internal sleeve shoulder 64 to halt the downward movement of the drop member 60. In this position, the 27 28 drop member 60, together with the diaphragm 80, 29 restrict or block the flow of mud down the drillstring. 30 When the mud-induced downforce on the drop member 60 31 becomes sufficiently high (with increased mud pumping, 32 if necessary), the shear pins 58 will rupture and thereby allow the sleeve 50 to be moved downwards from 33 its initial nozzle-blocking position as shown in Fig 3, 34 35 to a lower position in which the nozzles 44, 46, and 48

1 are opened for outflow of mud therethrough as 2 previously detailed. Next, the pressure of the mud on 3 the diaphragm 80 is increased (notwithstanding the now-commenced mud outflow through the nozzles 44-46), eg by increasing the working speed of the mud pumps 5 (not shown), to a level at which the diaphragm 80 bursts, thereby re-establishing mud flow down through 7 8 the stabiliser 10 to the drillbit. 9 The collective rupturing force of the shear pins 58 at 10 11 any given mud pressure will be selected to be less than the equivalent rupturing force of the diaphragm 80 at 12 the same mud pressure, ie it will be arranged that the 13 shear pins 58 will always rupture before the diaphragm 14 15 80 ruptures to ensure the intended sequence of 16 operations. 17 18 Mud flow down the drillstring is now split between the . 19 stabiliser nozzles 44-48 and the drillbit nozzles (not 20 shown) in proportion to their relative flow areas, with the greater portion of mudflow preferably going through 21 the drillbit to establish a substantial upflow of mud 22 23 past the exterior of the stabiliser 10 for removal of 24 debris arising from back-reaming operation of the 25 cutters 38-42. The lesser portion of mudflow goes through the nozzles 44-48 to provide the cooling, 26 27 lubrication, and cleaning functions previously detailed. 28 29 As the back-reaming operation takes place the 30 31 drillstring will be rotated at normal drilling speeds as it is slowly withdrawn from the wellbore. 32 33 cutters mounted on each stabiliser blade will cut and 34 remove the formation and hence permit the egress of the 35 BHA past the restriction.

The above example is cited as a simple, effective but 1 non-repeating means of back-reaming through a wellbore 2 3 restriction. By adaption of the mechanism used to divert the mud flow through the stabiliser nozzles, it 4 would be possible to have a back-reaming stabiliser 5 6 capable of back-reaming through a number of 7 restrictions in the wellbore. This repeatability could 8 be engendered by the use of sleeves or special valves 9 in the stabiliser body; valves which may be activated by either hydraulic or mechanical means which would 10 return to a detente position when the means of 11 12 activation was removed. 13 Whichever flow diverting valve means is used it is 14 15 preferred that the through bore of the stabiliser is 16 such that fishing and survey equipment may pass through 17 it without impinging on the valve operating mechanism 18 or snagging on any shoulders or ledges. 19 While certain modifications and variations have been 20 21 described above, the invention is not restricted 22 thereto, and other modifications and variations can be 23 adopted without departing from the scope of the 24 invention as defined in the appended claims. 25

Claims

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3 1. A back-reaming stabiliser comprising a tubular 4 body from which a plurality of stabiliser blades 5 extend, said tubular body having a through passage 6 for hydraulic fluid to flow internally through 7 said tubular body between opposite ends of said stabiliser in use thereof, said stabiliser blades 8 extending radially outwards of said tubular body, 9 10 said stabiliser blades extending longitudinally at 11 least partially along said tubular body, said stabiliser blades extending circumferentially at 12 13 least partially around said tubular body, radially 14 outer edges of said stabiliser blades being formed 15 as wellbore-bearing surfaces to provide a radially 16 supportive function in use of said stabiliser, 17 longitudinally common end edges of said stabiliser blades being formed as back-reaming cutter means, 18 19 and fluid vent means coupling said through passage to the exterior of said stabiliser for the flow 20 21 therethrough of hydraulic fluid from said through 22 passage to said cutter means, said back-reaming stabiliser being characterised by further 23 24 comprising valve means normally closing said fluid 25 vent means to the flow of hydraulic fluid therethrough, said valve means being selectively 26 27 operable during downhole use of said stabiliser to 28 open said fluid vent means to the flow of hydraulic fluid therethrough. 29

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2. A back-reaming stabiliser as claimed in Claim 1, wherein said valve means comprises a sleeve means movable in said through passage between a first position of said sleeve means in which said sleeve means closes said fluid vent to the passage of

1 hydraulic fluid therethrough, and a second 2 position of said sleeve means in which said sleeve 3 means opens said fluid vent means to the passage 4 of hydraulic fluid therethrough, said sleeve means 5 being movable from said first position thereof to said second position thereof by the application of 6 a substantial pressure differential between 7 longitudinally opposite ends thereof. 8

9

A back-reaming stabiliser as claimed in Claim 2, 10 3. wherein said sleeve means is normally retained in 11 said first position thereof by shear means 12 13 rupturable by said application of said substantial 14 pressure differential between longitudinally 15 opposite ends of said sleeve means thereupon to 16 release said sleeve means from said first position 17 thereof for movement to said second position 18 thereof.

19 20

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4. A back-reaming stabiliser as claimed in Claim 2 or Claim 3, wherein movement of said sleeve means from said first position thereof to said second position thereof is by way of sliding movement longitudinally along said through passage.

24 25

A back-reaming stabiliser as claimed in Claim 2 or 26 5. 27 Claim 3 or Claim 4, wherein said sleeve means is 28 formed with an internal passage extending 29 longitudinally therethrough and locally defining 30 said through passage of said stabiliser, said substantial pressure differential being created by 31 causing an at least temporary blockage of flow of 32 33 hydraulic fluid through said internal passage extending longitudinally through said sleeve 34 35 means.

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1 6. A back-reaming stabiliser as claimed in Claim 5, 2 wherein said hydraulic flow blockage is caused by manoeuvring a flow-blockage-inducing means into 3 4 said internal passage through said sleeve means 5 and said flow-blockage-inducing means comprises a drop means introduced to said stabiliser by being 6 7 dropped into the bore of the drillstring of which 8 the stabiliser forms part.

9

7. A back-reaming stabiliser as claimed in Claim 6
wherein said drop means comprises a generally
tubular member having a longitudinal through bore
initially blocked to the passage of hydraulic
fluid therethrough by a rupturable diaphragm
extending thereacross, said tubular member being
formed to lodge in said sleeve means.

17

18 8. A back-reaming stabiliser as claimed in Claim 7
19 wherein said tubular member is formed with an
20 external shoulder dimensioned to engage positively
21 with said sleeve means.

22

9. A back-reaming stabiliser as claimed in any preceding claim wherein said longitudinally common end edges of said stabiliser are formed as said back-reaming cutter means by embedding a plurality of hard inserts in leading portions of said edges.

28

29 10. A back-reaming stabiliser as claimed in Claim 9
30 wherein said inserts are located so as to lie
31 substantially in a common notional surface of
32 revolution.

33

11. A back-reaming stabiliser as claimed in Claim 1035 wherein said notional surface of revolution is

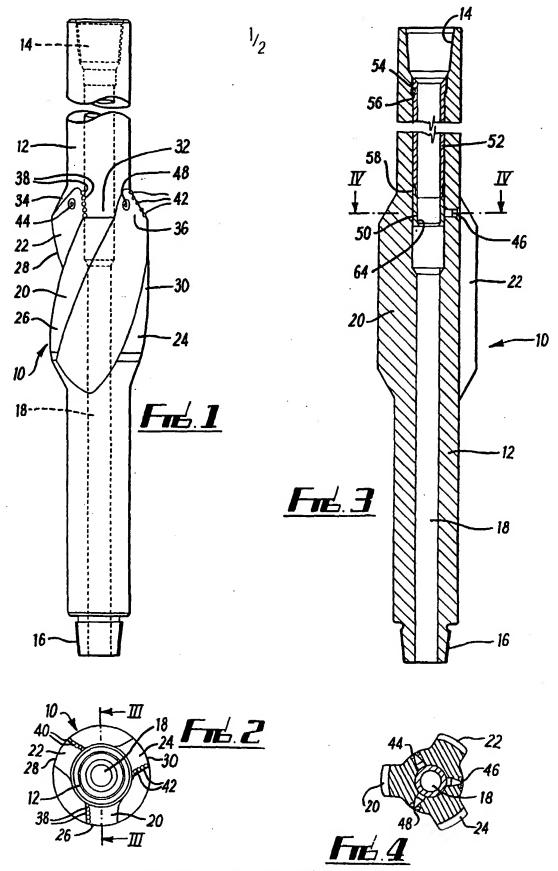
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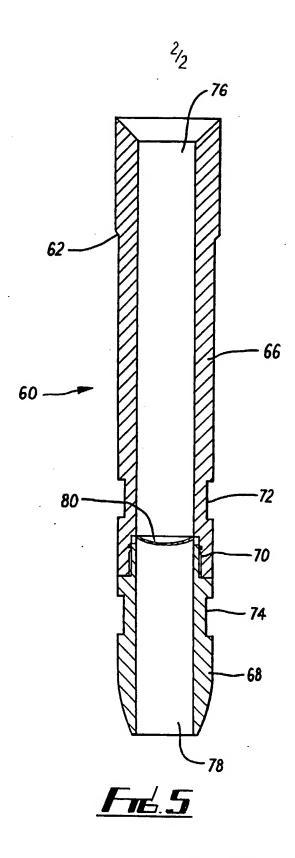
1	•	conical, and convergent in the longitudinal
2		direction of back-reaming operation.
3		
4	12.	A back-reaming stabiliser as claimed in any
5		preceding claim wherein said fluid vent means
6		comprises a respective fluid nozzle adjacent each
7		said cutter-formed end edge of said stabiliser
8		blades.
9		
LO	13.	A back-reaming stabiliser as claimed in any
Ll		preceding claim wherein said valve means is
L2		adapted for controllable cyclic operation
L3		repeatedly to open and close said fluid vent means
L 4		to the flow of hydraulic fluid therethrough at
15		selected times whereby said stabiliser may
L6		undertake plural episodes of back-reaming
L7		operation between which episodes said valve means
L8		re-closes said fluid vent means to the flow of
١9		hydraulic fluid therethrough.
20		•
21	14.	A back-reaming stabiliser as claimed in claim 13
22		wherein said valve means is adapted to open upon
23		the application thereto of a valvo-opening force

and to close automatically upon cessation of said

valve-opening force.



SUBSTITUTE SHEET



SUBSTITUTE SHEET

International Application N

L CLASSIFICATION OF SUBJECT MATTER (if several classification symbols apply, indicate all) ⁶							
According to International Patent Classification (IPC) or to both National Classification and IPC							
	5 E21B10/2 E21B21/1	6; E21B10/60;		1B34/14			
n. Fields	II. FTELDS SEARCHED						
		Minimum Docu	mentation Searched ⁷				
Classificati	ion System		Classification Symbols				
Int.Cl.	. 5	E21B					
			er than Minimum Documentation ts are Included in the Fields Searched ⁸				
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		D TO BE RELEVANT ⁹					
Category °	Citation of D	ocument, 11 with indication, where appro	priate, of the relevant passages 12	Relevant to Claim No. ¹³			
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A	21 Octo	618 010 (FALGOUT) ber 1986 whole document		1,9-12			
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A	21 Sept	981 360 (MARATHE) ember 1976 tract; figures 2,3		1-4			
			-/				
"T" later document published after the international filing date or priority date and not in conflict with the application but cited to understand the principle or theory underlying the invention. "E" earlier document but published on or after the international filing date. "L" document which may throw doubts on priority claim(s) or which is cited to establish the publication date of another citation or other special reason (as specified) "O" document referring to an oral disclosure, use, exhibition or other means "P" document published prior to the international filing date but later than the priority date claimed "A" document published after the international filing date or priority date and not in conflict with the application but cited to understand the priorityle or theory underlying the invention. "X" document of particular relevance; the claimed invention cannot be considered to involve an inventive step when the document is combined with one or more other such documents, such combination being obvious to a person skilled in the art. "A" document published after the international filing date or priority date and not in conflict with the application but cited to understand the principle or theory underlying the invention. "X" document of particular relevance; the claimed invention cannot be considered to involve an inventive step when the document is combined with one or more other such documents, such combination being obvious to a person skilled in the art.				he application but y underlying the Imed invention considered to imed invention tive step when the other such docu- o a person skilled			
IV. CERT	IFICATION						
Date of the	•	the International Search BER 1993	Date of Malling of this International Sea	1 1. 11. 93			
Internation	al Searching Authority EUROPE	AN PATENT OFFICE	Signature of Authorized Officer Héctor Fonseca				

III. DUCUME	NTS CONSIDERED TO BE RELEVANT (CONTINUED FROM THE SECOND SHEET)	
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ANNEX TO THE INTERNATIONAL SEARCH REPORT ON INTERNATIONAL PATENT APPLICATION NO.

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